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## A Multi-Mesh Finite Element Method for Phase Field Simulations

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# A Multi-Mesh Finite Element Method for Phase Field Simulations

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**Abstract.** In phase field models, different components of the solution (temperature and phase variable) exhibit a strongly different local behaviour. An optimal discretization should use a mesh for each component. We present a general framework for the adaptive solution of coupled systems and its application to phase field simulations.

Free boundary problems give rise to coupled systems of partial differential equations, which are prominent examples of systems where a careful numerical discretization is needed in order to resolve the solution's behaviour. Usually, a high resolution is essential near the free boundary, while coarser meshes are sufficient in the bulk. A reasonable numerical method should use locally adapted meshes to fulfil both the needs of accuracy and efficiency. For coupled systems like the phase field system, where components of the solution show strongly different local behaviour, an optimal discretization should use specially adapted meshes for each component.

## 1 Models for solidification

An undercooling or oversaturation of a liquid leads to a rapid solidification of the material. Models for this behaviour include energy and/or mass transport by diffusion and/or convection as well as energy/mass conservation conditions across the interface, like the Stefan condition. On a certain (meso-) scale, additional surface effects at the phase boundary play an important role, depending on curvature  $C$  or velocity  $V$  of the moving interface  $\Gamma$ , usually described by a Gibbs-Thomson relation like

$$\varepsilon_c C_\Gamma + \varepsilon_v V_\Gamma + \theta = 0 \quad \text{on } \Gamma.$$

Anisotropic surface effects lead to dendritic growth.

Mathematical models for solidification with surface effects differ mainly in the treatment of the free boundary, defining it as a sharp interface, a level set, or a diffuse interface (phase field). Numerical methods for solidification simulations are developed following the same lines.

### 1.1 Sharp interface models

In a sharp interface model, the phase boundary is a (smooth) hyper-surface of the underlying space (a surface in 3D or a curve in 2D). The advantage of the treatment of the interface as a smooth surface is its lower dimension and that curvature, which appears in the Gibbs-Thomson law, is well defined. Drawbacks are that the standard model allows no changes in interface topology, no nucleation. The notion of varifolds allows for some possibilities in this direction.

The free boundary problem leads to a degenerate parabolic equation for the interface motion, like an anisotropic mean curvature flow equation, coupled with heat or mass diffusion in solid and liquid phases. Additionally, convection can be added by coupling to Navier-Stokes' equations in the time dependent liquid phase.

**Numerical methods for sharp interface simulations.** The sharp interface model leads to numerical methods with separate discretizations of bulk (3D) and interface (2D) (or 2D & 1D).

In the bulk, a fine mesh is essential near the interface to sufficiently resolve the behaviour of solutions (temperature, concentration). Adaptively refined meshes in finite element methods are crucial to get enough resolution, especially in 3D. A numerical sharp interface finite element method with adaptive meshing in 2D and 3D was presented in [7,8], with additional convection in 2D in [2].

### 1.2 Diffuse interface models

Phase field models can be formulated either with a double obstacle potential or a smooth double well potential. We want to restrict ourself here to an obstacle formulation, as introduced by Blowey and Elliott [3].

The model describes the evolution of temperature  $\theta$  and phase variable  $\chi$  in a domain  $\Omega$  and is a system of two (degenerate) parabolic equations:

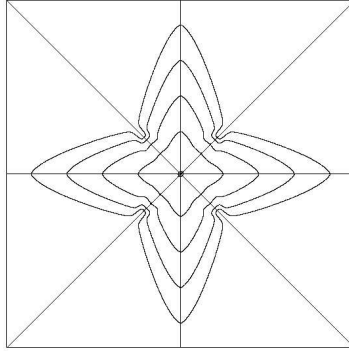
$$\begin{aligned} \partial_t(\theta + \lambda\chi) - \kappa\Delta\theta &= f \\ \varepsilon\partial_t\chi - \varepsilon\operatorname{div}(a(\nabla\chi)) + \Lambda(\chi) - \frac{1}{\varepsilon}\beta\chi &\ni \gamma\theta \end{aligned}$$

in  $\Omega \times (0, T)$  plus boundary and initial conditions. The function  $a$  may include anisotropic solidification parameters, and  $\Lambda$  is a (set valued) maximal monotone graph, the subdifferential of the double obstacle potential,

$$\Lambda(s) = \begin{cases} (-\infty, 0] & \text{if } s = -1 \\ 0 & \text{if } s \in (-1, 1) \\ [0, +\infty) & \text{if } s = +1 \end{cases}$$

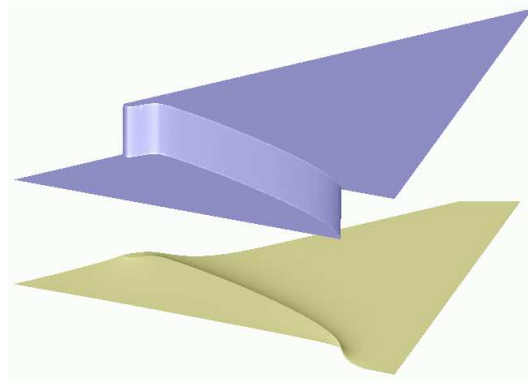
with the effect that values of  $\chi$  are in the interval  $[-1, +1]$ .  $f$  is a given heat source density,  $\lambda$ ,  $\kappa$ , and  $\beta$  are non-negative, material-dependent coefficients,

and  $\varepsilon > 0$  a small parameter. The phase variable  $\chi$  is equal to  $-1$  (solid) or  $+1$  (liquid) everywhere but in a narrow transition region of width  $O(\varepsilon)$ . For  $\varepsilon \rightarrow 0$ , the solution converges against a solution of the sharp interface model with a corresponding Gibbs-Thomson relation. Figure 1 shows the evolution of the phase boundary during the solidification of an undercooled liquid.



**Fig. 1.** Solid-liquid interface at different times

Both components of the solution, temperature and phase variable, show a strongly different local behaviour, see Figure 2.



**Fig. 2.** Graphs of phase variable and temperature on  $1/8$  domain

- The phase variable is constant outside of a moving narrow strip of width  $O(\varepsilon)$ , where the phase transition occurs. Here,  $|\nabla\chi| = O(\varepsilon^{-1})$ .
- The temperature satisfies the heat equation outside this strip, thus it is smooth. Inside the moving strip, the gradient of temperature changes

rapidly. In the limit  $\varepsilon \rightarrow 0$ , the Stefan condition holds at the sharp interface,

$$(\kappa \nabla \theta_{\text{liquid}} - \kappa \nabla \theta_{\text{solid}}) \cdot n = -\lambda V.$$

In order to resolve this behaviour with a finite element approximation, different requirements to the discretization hold for both components.

- The mesh for discretization of  $\chi$  must have a local mesh width of  $h_S < c\varepsilon$ . Outside this strip, where  $\chi$  is constant, the mesh might be arbitrarily coarse. In order to be able to track nucleations, the phase field system should be solved in the whole domain, not only near the current interface.
- To resolve the temperature behaviour, a much coarser mesh is sufficient in the strip than is needed for the phase variable. On the other hand, the mesh must have a sufficient fineness also in the rest of the domain.

A separate discretization for both temperature and phase variable is needed in order to meet all requirements in an efficient numerical method.

## 2 Adaptive finite element methods for coupled systems of PDE

Besides phase transitions, many physical problems lead to coupled systems of partial differential equations, too. Especially in case of nonlinear phenomena, the components  $u_i$ ,  $i = 1, \dots, n$  may show a strongly different behaviour (smoothness of solutions etc.) in the common underlying domain  $\Omega \subset \mathbb{R}^d$ .

The usual adaptive discretization for systems of PDEs uses the same mesh for all components, locally refined based on a posteriori error indicators for the sum of error contributions on each mesh element.

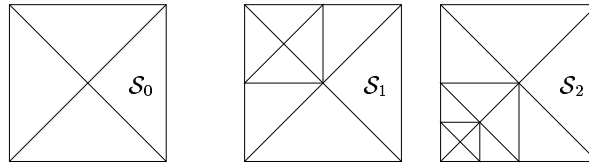
An optimal adaptive discretization should use different locally refined meshes for different components of the solution, controlled by separate error indicators for each component.

In [6] we present a general concept for adaptive finite element methods for stationary or time dependent coupled problems.

Solution components  $u_i$  are discretized in finite element spaces  $X_i$ , which are based on different locally refined simplicial grids  $\mathcal{S}_i$ , where all meshes  $\mathcal{S}_i$  are refinements of the same macro triangulation  $\mathcal{S}_0$  of  $\Omega$ , see Figure 3 for a simple example.

The advantages of this approach are

- separately adapted mesh for each component of the solution;
- altogether less degrees of freedom, thus more efficient;
- by the common macro mesh, a direct local hierarchy of the triangulations  $\mathcal{S}_i$  is given. This enables an *exact* evaluation of each other component  $U_j$  on the elements of mesh  $\mathcal{S}_i$ .



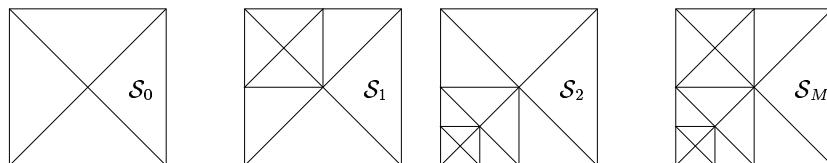
**Fig. 3.** Two different refinements of a macro triangulation  $\mathcal{S}_0$

A somewhat similar idea for phase field simulations was proposed by Elliott and Gardiner [5], who use two separate meshes for temperature and phase variable. A fine mesh with mesh size  $h \ll \varepsilon$  for the phase variable is used only near the transition region (selected by a ‘mask’), a coarse mesh with mesh size  $H = 4h$  for the temperature. Use of the ‘mask’ reduces computations for the phase variable significantly, but does not allow to track nucleations, e. g. Both meshes are aligned, but no error estimators or local mesh refinements are used.

## 2.1 Aspects of implementation

The adaptive multi-mesh method is implemented in the finite element toolbox ALBERT, a joint development with K. G. Siebert [9,10]. The toolbox uses simplicial meshes in 2D (triangles) and 3D (tetrahedra) and local refinement by bisection of elements, which induces a hierarchical structure of meshes and finite element spaces.

The multi-mesh concept introduces the traversal of a common, *virtually refined* mesh  $\mathcal{S}_M$ , see Figure 4, for calculation of coupling terms, which involve several components or functions from finite element spaces defined on different meshes, like  $\int_{\Omega} U_i \Phi$  with  $U_i \in X_i$ ,  $\Phi \in X_j$ .



**Fig. 4.** Virtual mesh  $\mathcal{S}_M$  with locally maximal refinement from  $\mathcal{S}_1, \mathcal{S}_2$

## 2.2 Numerical analysis and adaptive methods

We recall the standard a posteriori error estimates and adaptive finite element methods for a scalar elliptic problem. The error  $\|u - U\|$  between the solution  $u$

and a discrete approximation  $U$  is estimated by a sum of local error indicators, e. g.

$$\|u - U\|^2 \leq \sum_{S \in \mathcal{S}} \eta_S^2,$$

where the indicators  $\eta_S$  must be computable from the discrete solution  $U$  and given data of the problem (*a posteriori*).

On an quasi-optimal mesh for a given error tolerance  $tol$ , the local error indicators are equally distributed over all mesh elements,

$$\eta_S^2 \approx \frac{tol^2}{\#\mathcal{S}} \quad \text{for all } S \in \mathcal{S},$$

compare [1], e. g. A similar approach is used in case of coupled systems. The goal is an estimate of the errors  $\|u_i - U_i\|$  by a sum of local error indicators on separate meshes, like

$$\sum_{i=1}^n \|u_i - U_i\|^2 \leq \sum_{S \in \mathcal{S}_1} \eta_{1,S}^2 + \cdots + \sum_{S \in \mathcal{S}_n} \eta_{n,S}^2.$$

In order to describe a quasi-optimal set of meshes, one possibility is to split the total tolerance  $tol$  to the separate meshes

$$tol_1^2 + \cdots + tol_n^2 = tol^2$$

and optimize every single mesh. This leads to decoupled equidistribution conditions

$$\eta_{i,S}^2 \approx \frac{tol_i^2}{\#\mathcal{S}_i} \quad \text{for all } S \in \mathcal{S}_i, i = 1, \dots, n.$$

### 3 Adaptive method for phase field models

In a joint paper with Z. Chen and R.H. Nochetto [4], we derive error estimates and adaptive methods for the double obstacle phase field system. Denoting by  $u = \theta + \lambda\chi$  the energy density, and again discrete (finite element) functions by uppercase letters, the estimate looks like

$$\begin{aligned} & \|u - U\|_{L^\infty(0,T;H^{-1}(\Omega))} + \sqrt{\varepsilon} \|\chi - X\|_{L^\infty(0,T;L^2(\Omega))} \\ & + \left( \int_0^T \|\theta - \Theta\|_{L^2(\Omega)}^2 + \alpha\varepsilon \|\nabla\chi - \nabla X\|_{L^2(\Omega)}^2 dt \right)^{1/2} \\ & \leq \text{computable terms, localizable to mesh elements.} \end{aligned}$$

In [4], we use a common mesh for the discretization of temperature and phase variable. The local error indicators  $\eta_S$  contain contributions from both components,

$$\text{Estimated error} \leq \eta_0 + \max_m \sum_{S \in \mathcal{S}^m} \eta_S^2,$$



where  $m$  is the time step index. A quasi-optimal common mesh fulfils the equidistribution condition for the local indicators

$$\eta_S^2 \approx \frac{tol^2}{\#\mathcal{S}^m} \quad \text{for all } S \in \mathcal{S}^m.$$

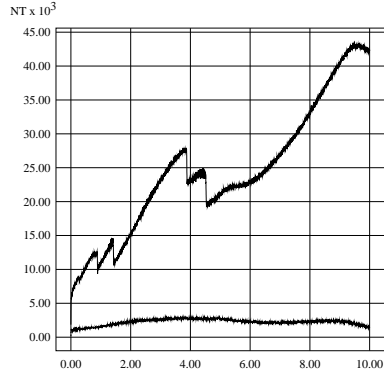
Here, the total mesh element count  $\#\mathcal{S}^m$  is very large because of the fine resolution in the strip, thus the local temperature error must be very small also in elements far from the interface.

### Adaptive FE method with separate meshes

A separation of contributions from temperature and phase variable in the error indicator is possible:

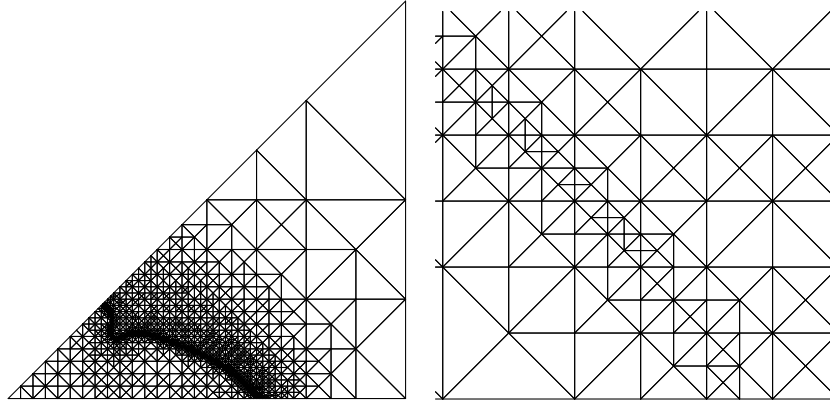
$$\text{Estimated error} \leq \eta_0 + \max_m \left( \sum_{S \in \mathcal{S}_\theta^m} \eta_{\theta,S}^2 + \sum_{S \in \mathcal{S}_\chi^m} \eta_{\chi,S}^2 \right).$$

A split of the tolerance to both meshes, as described above, leads to a smaller element count  $\#\mathcal{S}_\theta^m$ , thus larger local error tolerances for the temperature error. It follows that the temperature mesh contains much less elements than a common mesh. Also the mesh for the phase variable contains less elements, as a fine resolution is needed only in the transition region. Figure 5 shows element counts over time. A time-dependent tolerance was used for the error in each time step, which reflects the fact that the interface length grows during the simulation.

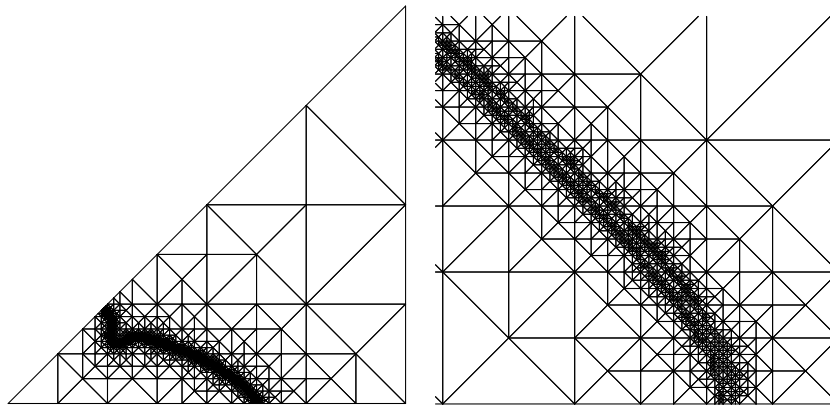


**Fig. 5.** Element counts for phase variable and temperature meshes over time

Figures 6 and 7 show the temperature and phase variable meshes with zooms to the interface region. Finally, we present in Figures 8-10 a comparison of temperature and phase variable meshes from simulations with three different error tolerances  $tol$ .



**Fig. 6.** Temperature mesh (2608 elements) and zoom to interface



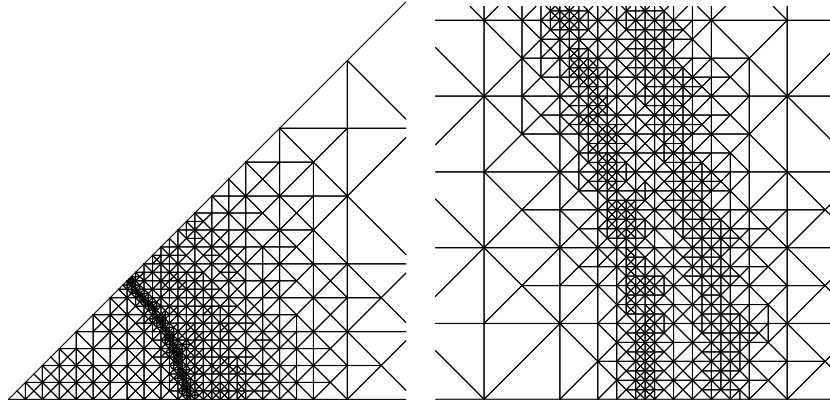
**Fig. 7.** Phase variable mesh (23015 elements) and zoom

## Conclusion

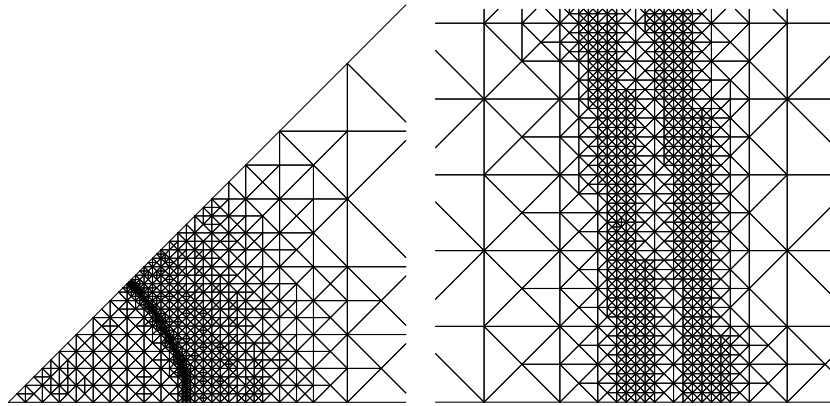
We described an efficient adaptive finite element method for phase field calculations, together with some 2D simulation results. A more detailed description of the multi-mesh method and results from 3D simulations will be presented in forthcoming articles.

## Acknowledgements.

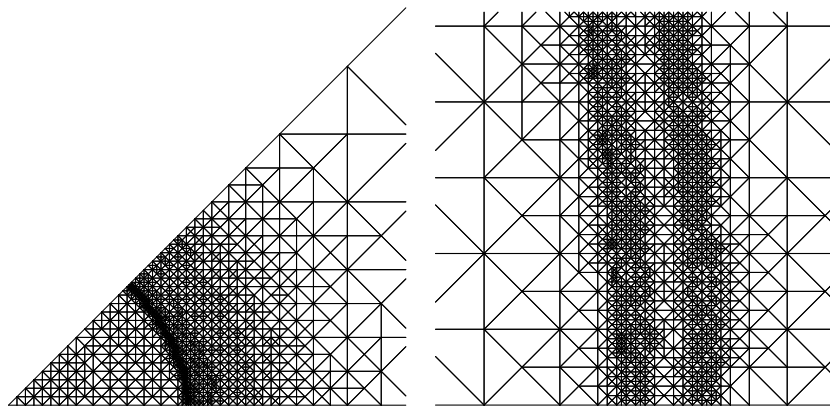
This article reports partly joint work with Z. Chen (Beijing), R. H. Nochetto (College Park), and K. G. Siebert (Freiburg).



**Fig. 8.**  $tol=10$ : Meshes with 1122 resp. 7440 elements



**Fig. 9.**  $tol=7$ : Meshes with 1568 resp. 11813 elements



**Fig. 10.**  $tol=5$ : Meshes with 2245 resp. 20631 elements

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